

16 March 1979

STATINTL

Square Radome
Structural Engineering
Handbook
p 19-16 thru 19-17

WINDLOADING REQUIREMENTS

1. The windload from 150 mph wind is computed as follows:

$$\begin{aligned} \text{psf} &= .00256V^2 & V &= \text{mph} \\ &= .00256(150)^2 \\ &= 57.6 \text{ psf} \end{aligned}$$

2. The windload on the shed with 2 panels removed and a 40 mph wind. The positive pressure is assumed to be at full load while the negative pressure, due to a "vacuum" on the leeward side is 0.40.

$$\begin{aligned} \text{psf} &= .00256V^2 (\text{positive factor} + \text{negative factor}) \\ &= .00256(40)^2 (1.00 + 0.40) \\ &= (.00256)(1600)(1.4) \\ &= 5.7344 \text{ psf} \end{aligned}$$

3. A 60 mph wind with 30 psf load from snow or ice

$$\begin{aligned} \text{psf} &= (.00256)(V^2) + 30 \\ &= (.00256)(3600) + 30 \\ &= 9.216 + 30 \\ &= 39.216 \text{ psf} \end{aligned}$$

4. The requirement to resist 70 psf with no wind appears to be the worst case for the roof. If this load is ice, the ice would have to be

$$\frac{(70)(12)}{(62.4)(.88)} = 15 \text{ inches thick}$$

If the load is snow, the depth would be

$$\frac{(70)(12)}{(62.4)(.125)} = 107.7 \text{ inches or about 9 feet deep}$$

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REF. DCE-MTB-535

DESIGN AN ADAPTER SO THAT EXISTING FOUNDATION
BOLTS CAN BE USED WITH A NEWER MODEL ANTENNA

FURNISHED ARE

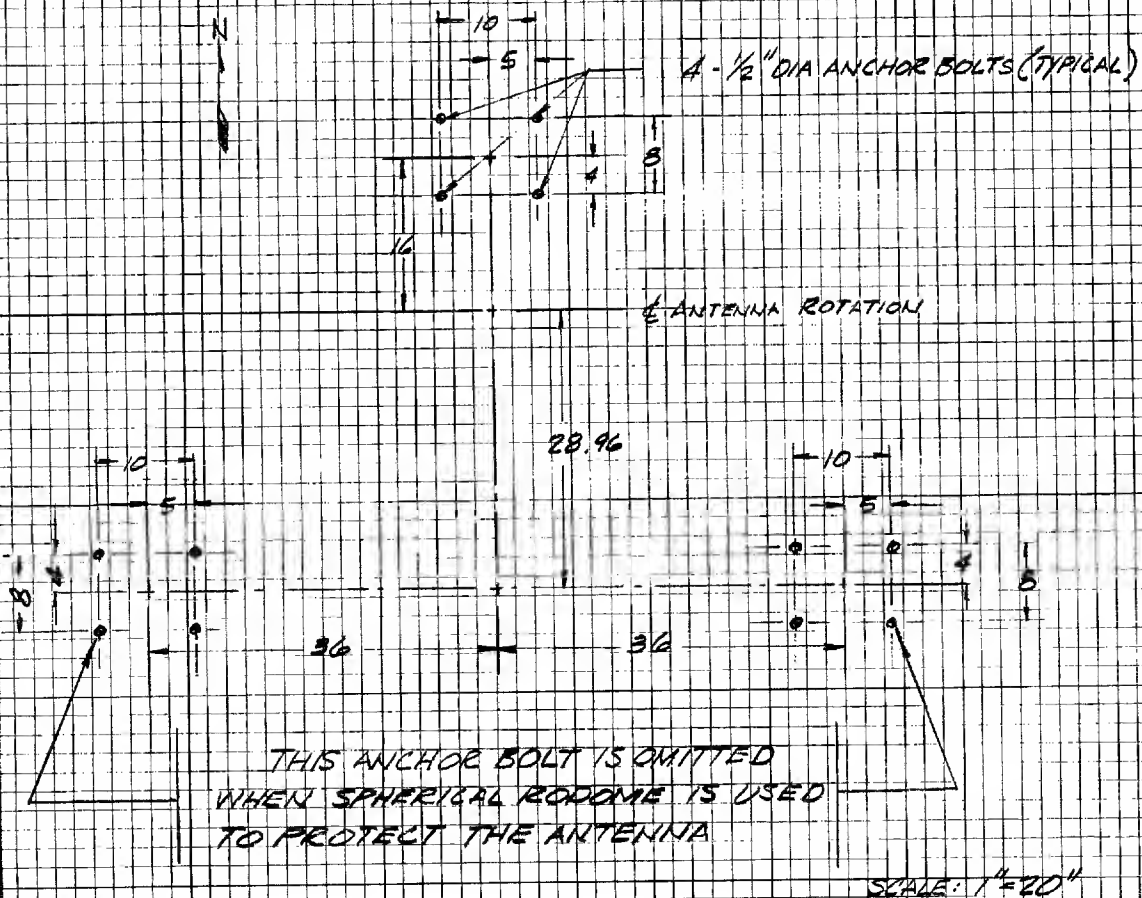
OLD ANTENNA BOLT LAYOUT

NEW ANTENNA BOLT LAYOUT

CLEARANCES OF NEW ANTENNA IN TWO STYLES OF RADOME
CALCULATED WIND LOADS ON NEW ANTENNA.

FURNISH PLANS AND SPECIFICATIONS TO CONSTRUCT ADAPTER

EXISTING ANTENNA BOLT LAYOUT



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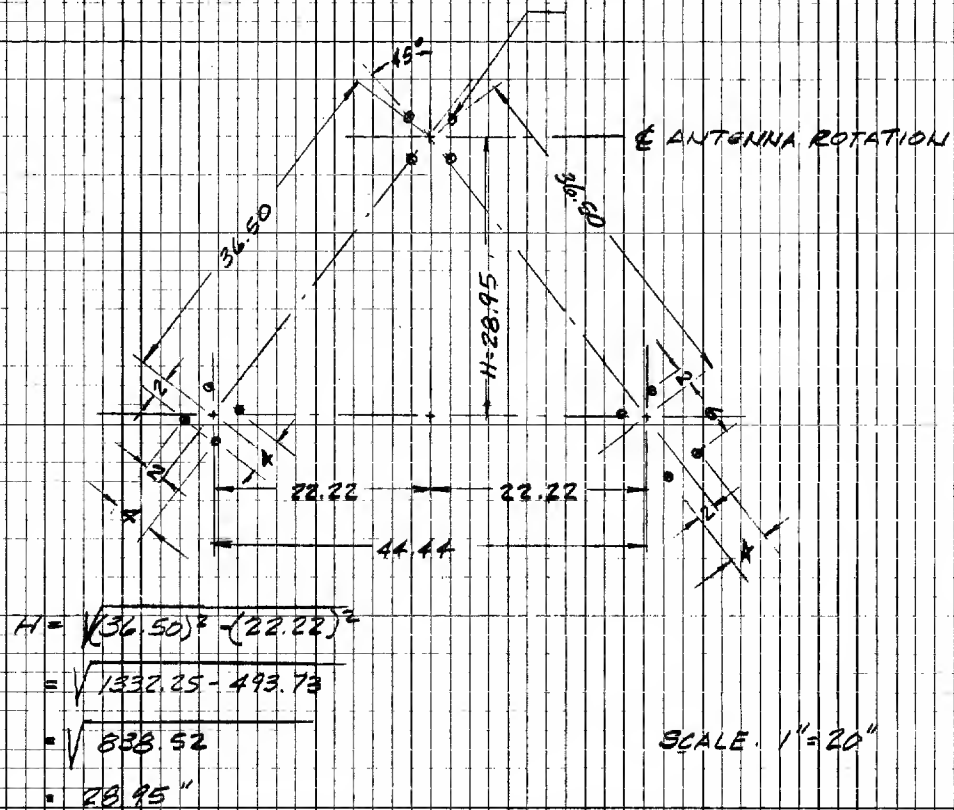
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NEW ANTENNA BOLT LAYOUT

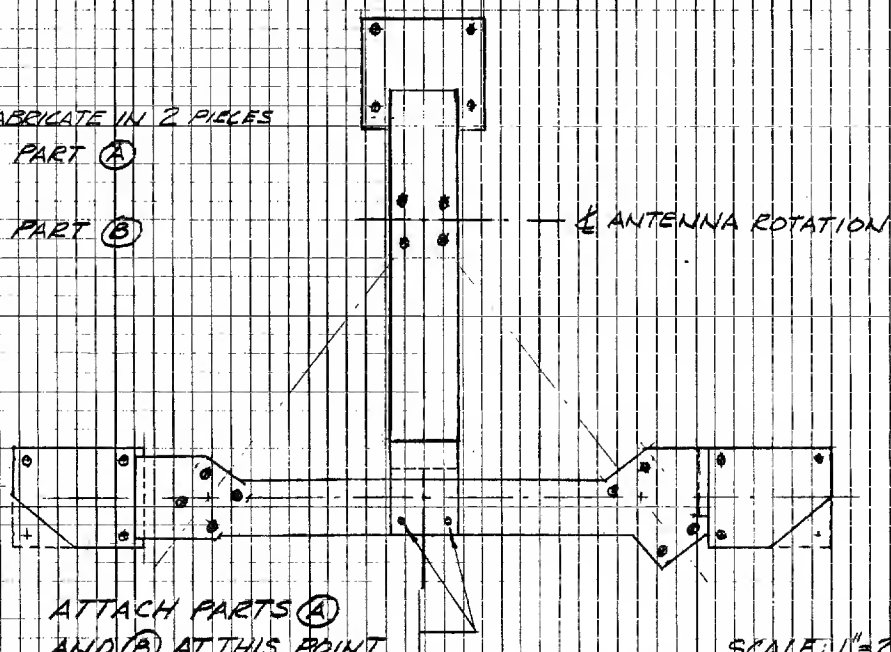


ANTENNA ADAPTER BOLT LAYOUT COMBINING OLD AND NEW REQUIREMENTS

FABRICATE IN 2 PIECES

PART (A)

PART (B)



ATTACH PARTS (A)
AND (B) AT THIS POINT
WITH 3 - 1/2" DIA BOLTS

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LOAD CALCULATIONS

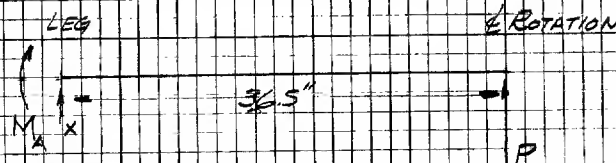
THE WEIGHT OF THE ANTENNA IS CONCENTRATED DOWNWARD AT THE CENTER OF THE POINT OF ROTATION. OF A TOTAL WEIGHT OF 800 POUNDS, ASSUME 700 POUNDS ACTS ON THE ADAPTER AT THIS POINT UNDER A "NO WIND" CONDITION.

FROM PREVIOUS CALCULATIONS FOR WIND LOADS

150 mph yields 5405 lbs/ft of moment

100 mph yields 2400 lbs/ft. of moment

USING THE MOMENT ARM OF 36.5 INCHES, THE FOLLOWING ADDITIONAL FORCE WOULD BE ADDED TO THE ADAPTER PART (A) AT THE POINT OF ROTATION.



150 mph

$$\sum M_x = 0$$

$$5405 = \frac{36.5}{12} P_{150}$$

$$P_{150} = \frac{(5405)(12)}{36.5}$$

$$P_{150} = 1777 \text{ lbs.}$$

100 mph

$$\sum M_x = 0$$

$$2400 = \frac{36.5}{12} P_{100}$$

$$P_{100} = \frac{(2400)(12)}{36.5}$$

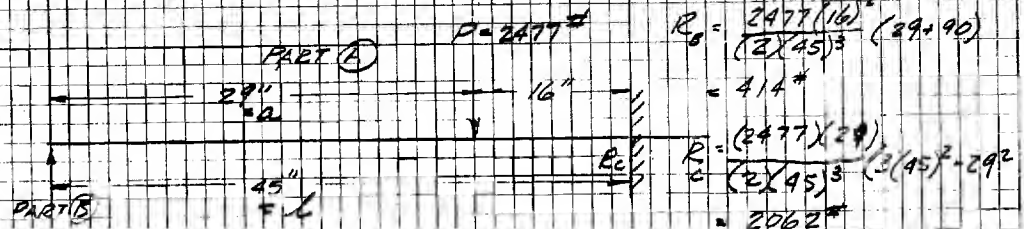
$$P_{100} = 789 \text{ lbs}$$

TOTAL FORCE AT P

$$\begin{array}{r} 1777 \\ 700 \\ \hline 2477 \end{array}$$

$$\begin{array}{r} 789 \\ 700 \\ \hline 1489 \end{array}$$

FOR PART (A) ASSUME THAT THE END ATTACHED TO PART (B) IS FREE TO ROTATE WHILE THE END ATTACHED TO THE OLD FOUNDATION BY 4 EXISTING BOLTS IS FIXED OR MOMENT RESISTANT.



$$P = 2477 \text{ lbs}$$

$$R_b = \frac{2477(16)^2}{(2)(45)^3} (29 + 40)$$

$$= 414 \text{ lbs}$$

$$R_c = \frac{(2477)(29)}{(2)(45)^3} (3(45)^2 - 29^2)$$

$$= 2062 \text{ lbs}$$

THE MAXIMUM DEFLECTION, Δ_{max} , IS THE REPRESENTED BY

$$\Delta_{max} = \frac{Pa(L^2 - a^2)^2}{3EI(3L^2 - a^2)^2}$$

AND $E = 30 \times 10^6$ FOR STEEL

I = MOMENT OF INERTIA OF SECTION

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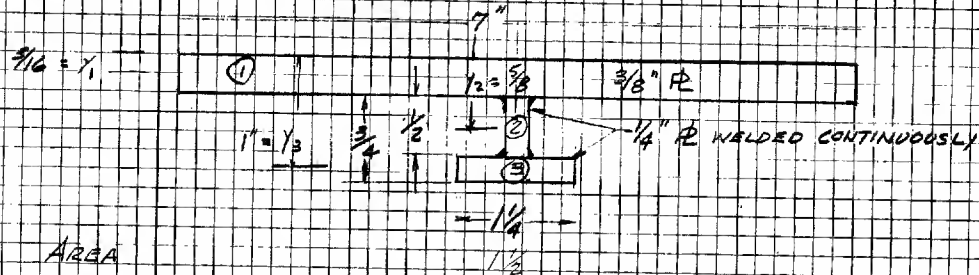
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FROM ADAPTER SKETCH ON PAGE 2, PART (A) IS APPROXIMATELY 7" WIDE AT THE AREA WHERE BENDING WILL OCCUR. USE THE FOLLOWING STIFFENED SECTION.



AREA

$$\begin{aligned}
 A &= A_1 + A_2 + A_3 \\
 &= (7) \left(\frac{3}{8}\right) + \left(\frac{1}{2}\right) \left(\frac{1}{4}\right) + \left(\frac{1}{4}\right) \left(\frac{1}{4}\right) \\
 &= 2.625 + 0.125 + 0.3125 \\
 &= 3.0625 \text{ in}^2
 \end{aligned}$$

CENTROID - \bar{y}

$$\begin{aligned}
 \sum A y &= A_1 y_1 + A_2 y_2 + A_3 y_3 \\
 3.0625 \bar{y} &= (2.625)(0.1875) + (0.125)(0.625) + (0.3125)(1.00) \\
 3.0625 \bar{y} &= 0.4922 + 0.0781 + 0.3125 \\
 \bar{y} &= \frac{0.8828}{3.0625} \\
 \bar{y} &= 0.2883"
 \end{aligned}$$

MOMENT OF INERTIA - \bar{I}

$$\begin{aligned}
 \bar{I} &= I_1 + A_1 (\bar{y} - y_1)^2 + I_2 + A_2 (\bar{y} - y_2)^2 + I_3 + A_3 (\bar{y} - y_3)^2 \\
 &= \frac{1}{12} (7) \left(\frac{3}{8}\right)^3 + 2.625 (0.2883 - 0.1875)^2 + \frac{1}{12} \left(\frac{1}{4}\right) \left(\frac{1}{2}\right)^3 + 0.125 (0.625 - 0.2883)^2 \\
 &\quad + \frac{1}{12} \left(\frac{1}{4}\right) \left(\frac{1}{4}\right)^3 + 0.3125 (1.000 - 0.2883)^2 \\
 &= (.5833)(.0527) + 2.625 (.0102) + .0026 + 0.125 (.1134) \\
 &\quad + .1042 (.0313) + 0.3125 (.5065) \\
 &= .0307 + .0268 + .0026 + .0142 + .0033 + .1583 \\
 &= .2359
 \end{aligned}$$

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Option 2 INCREASE WIDTH OF BOTTOM MEMBER FROM $1/4"$ TO $1/2"$

AREA

$$A = A_1 + A_2 + A_3$$

$$A = 2.625 + 0.125 + 0.375$$

$$A = 3.125$$

CENTROID - \bar{y}

$$\Sigma A\bar{y} = A_1\bar{y}_1 + A_2\bar{y}_2 + A_3\bar{y}_3$$

$$3.125\bar{y} = 0.4922 + 0.0781 + 0.375$$

$$\bar{y} = \frac{0.9453}{3.125}$$

$$\bar{y} = 0.302$$

MOMENT OF INERTIA - \bar{I}

$$I = I_1 + A_1(\bar{y} - \bar{y}_1)^2 + I_2 + A_2(\bar{y} - \bar{y}_2)^2 + I_3 + A_3(\bar{y} - \bar{y}_3)^2$$

$$= .0307 + .0344 + .0026 + .0130 + .0039 + .1827$$

$$= .2673$$

DEFLECTIONS - Δ

$$\Delta = \frac{Pa(L^2 - a^2)^3}{36EI(3L^2 - a^2)^2}$$

$$= \frac{P}{I} \left[\frac{a(L^2 - a^2)^3}{36(3L^2 - a^2)^2} \right]$$

$$= \frac{P}{I} \left[\frac{29(45^2 - 29^2)^3}{(3)(30)(10^6)(3)(45^2 - 29^2)^2} \right]$$

$$= \frac{P}{I} \left[\frac{29(1184)^3}{(3)(30)(10^6)(5234)^2} \right]$$

$$= \frac{P}{I} \left[\frac{(29)(1.6598)(10^9)}{(3)(30)(10^6)(2.739)(10^4)} \right]$$

$$= \frac{P}{I} \cdot .00001953$$

$$\Delta_{150} = \frac{(2477)}{.2359} (.00001953) = .205 \text{ in}$$

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$$\Delta_{150 \text{ option}} = \frac{2477}{.2673} (.00001953)$$

$$= .181 \text{ in}$$

$$\Delta_{100} = \frac{1489}{.2359} (.00001953)$$

$$= .123$$

$$\Delta_{100 \text{ option}} = \frac{1489}{.2673} (.00001953)$$

$$= .109 \text{ in}$$

STRESS IN SECTION AT CONNECTION TO EXISTING NORTH ANCHORS

$$M = \frac{f I}{\epsilon}$$

$$f = \frac{M c}{I}$$

$$M_{\text{rotation}} = 414 \left(\frac{29}{12} \right)$$

$$= 1000 \text{ ft}^{\#}$$

$$M_{\text{fixed}} = \frac{(2477) \left(\frac{29}{12} \right) \left(\frac{16}{12} \right)}{2 \left(\frac{45}{12} \right)^2} \left(\frac{29}{12} + \frac{45}{12} \right)$$

$$= \frac{(2477)(2417)(1.333)}{(2)(14.062)} (6.167)$$

$$= 1749.97 \text{ ft}^{\#}$$

[1051 ft. # for 100 mph]

$$f = \frac{(1749.97)(12)(1.125 - 300)}{.2673}$$

$$= (1749.97)(12)(0.823)$$

$$= 62620 \text{ psi}$$

Since the upper limit of the most readily available steel, A-36, is 36,000 psi yield, the adapter must be stiffened at this point to insure no inelastic deformation during 150 mph wind condition.

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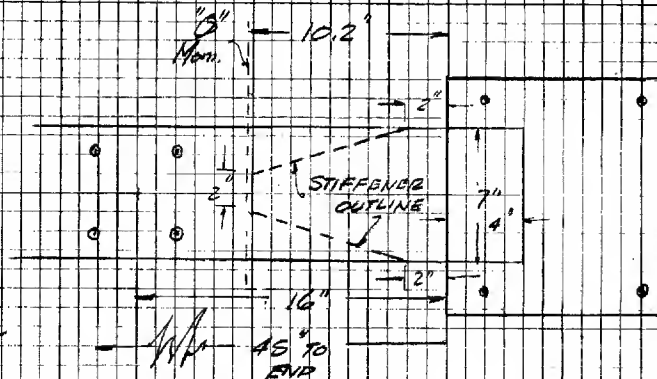
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STIFFENER FOR PART (A) AT CONNECTION TO EXISTING ANCHOR BOLTS



POINT OF O MOMENT

$$\begin{aligned}
 &= \frac{P_1}{R_2} \\
 &= \frac{(2477)(29")}{2062} \\
 &= 34.8" \text{ from } R_2 \quad \therefore 45" - 34.8" = 10.2"
 \end{aligned}$$

ASSUME STIFFENER TO BE $\frac{1}{2}$ " THICK EXTENSION OF SPACER BETWEEN THE
THE PLATES. THE EFFECTIVE CROSS SECTION WOULD THEN BE
7" WIDE AND $\frac{1}{8}$ " THICK

$$\begin{aligned}
 I &= \frac{1}{12} b h^3 \\
 &= \frac{1}{12} (7) \left(\frac{1}{8}\right)^3 \\
 &= 0.391
 \end{aligned}$$

$$\begin{aligned}
 f &= \frac{M c}{I} \\
 &= \frac{(1749)(12)\left(\frac{1}{16}\right)}{0.391}
 \end{aligned}$$

= 23,484 psi WHICH IS WITHIN THE 36,000 psi limit

WOULD STIFFENER BE REQUIRED UNDER 100mph CONDITIONS

$$\begin{aligned}
 f &= \frac{M c}{I} \\
 &= \frac{(1000)(12)\left(\frac{1}{16}\right)}{0.391}
 \end{aligned}$$

= 36,947 psi WHICH IS VERY CLOSE TO 36,000 psi limit.
STIFFENER NOT REQUIRED

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ESTIMATED INT.

PART (A)

$$\frac{9}{8} R$$

$$(12 \times 7) = 84$$

$$(40 \times 7) = 280$$

$$(12 \times 15) = \frac{156}{520 D''}$$

$$\frac{520}{144} = 3.61 D'' @ 15.3 \frac{1}{D''^2} = 55.23$$

$$\frac{1}{2} R$$

$$(4 \times 7) = 28$$

$$(2 \frac{1}{2} \times 8) = 20$$

$$(2 \times 8) = \frac{16}{78}$$

$$\frac{78}{144} = .54 D'' @ 20.4 \frac{1}{D''^2} = 11.02$$

$$\frac{1}{4} R$$

$$(\frac{1}{4} + \frac{1}{2}) (35) = 6.125$$

$$\frac{6.125}{144} = .0425 D'' @ 10.2 \frac{1}{D''^2} = 4.34$$

70.6 D''

PART (B)

$$\frac{1}{3} R$$

$$(14 \times 10 \frac{1}{3} \times 2) = 294$$

$$- (5 \times 7) = -35$$

$$(38 \times 6) = 228$$

$$(2 \frac{1}{2} \times 6) = 15$$

$$(6 \times 11) = 66$$

$$(5 \times \frac{8}{2}) = 20$$

$$(7 \times \frac{9}{2}) = 31.5$$

$$(7 \times 5) = 35$$

$$(5 \times \frac{5}{2}) = \frac{12.5}{66.7}$$

$$\frac{66.7}{144} = 4.63 D'' @ 15.3 \frac{1}{D''^2} = 70.84 D''$$

$$\frac{1}{2} R$$

$$(2 \times 7) = 14$$

$$(2 \times 8) = \frac{16}{30}$$

$$\frac{30}{144} = .21 D'' @ 20.4 \frac{1}{D''^2} = 4.25$$

79.43 D''

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<p>TOTAL WT.</p> <p>PART (A) = 70.6#</p> <p>PART (B) = 79.4#</p> <p>150.0# plus nuts, bolts, washers</p>				